# Dynamics of the Elastic Pendulum 

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## Agenda

- Introduction to the elastic pendulum problem
- Derivations of the equations of motion
- Real-life examples of an elastic pendulum
- Trivial cases \& equilibrium states
- MATLAB models


## The Elastic Problem (Simple Harmonic Motion)

- $F_{n e t}=m \frac{d^{2} x}{d t^{2}}=-k x \rightarrow \frac{d^{2} x}{d t^{2}}=-\left(\frac{k}{m}\right) x$
- Solve this differential equation to find

$$
x(t)=c_{1} \cos (\omega t)+c_{2} \sin (\omega t)=A \cos (\omega t-\varphi)
$$

- With velocity and acceleration

$$
\begin{aligned}
& v(t)=-A \omega \sin (\omega t+\varphi) \\
& a(t)=-A \omega^{2} \cos (\omega t+\varphi)
\end{aligned}
$$

- Total energy of the system

$$
\begin{aligned}
& E=K(t)+U(t) \\
& =\frac{1}{2} m v t^{2}+\frac{1}{2} k x^{2}=\frac{1}{2} k A^{2}
\end{aligned}
$$

## The Pendulum Problem (with some assumptions)

- With position vector of point mass $\vec{x}=l(\sin \theta \vec{\imath}-\cos \theta \vec{\jmath})$, define $\vec{r}$ such that $\vec{x}=l \vec{r}$ and $\vec{\theta}=\cos \theta \vec{\imath}+\sin \theta \vec{\jmath}$
- Find the first and second derivatives of the position vector:

$$
\begin{gathered}
\frac{d \vec{x}}{d t}=l \frac{d \theta}{d t} \vec{\theta} \\
\frac{d^{2} \vec{x}}{d t^{2}}=l \frac{d^{2} \theta}{d t^{2}} \vec{\theta}-l\left(\frac{d \theta}{d t}\right)^{2} \vec{r}
\end{gathered}
$$

- From Newton's Law, (neglecting frictional force)

$$
m \frac{d^{2} \vec{x}}{d t^{2}}=\overrightarrow{F_{g}}+\overrightarrow{F_{t}}
$$

## The Pendulum Problem (with some assumptions)

Defining force of gravity as $\overrightarrow{F_{g}}=-m g \vec{\jmath}=m g \cos \theta \vec{r}-$ $m g \sin \theta \vec{\theta}$ and tension of the string as $\overrightarrow{F_{t}}=-T \vec{r}$ :

$$
\begin{aligned}
-m l\left(\frac{d \theta}{d t}\right)^{2} & =m g \cos \theta-T \\
m l \frac{d^{2} \theta}{d t^{2}} & =-m g \sin \theta
\end{aligned}
$$

Define $\omega_{0}=\sqrt{g / l}$ to find the solution:

$$
\frac{d^{2} \theta}{d t^{2}}=-\frac{g}{l} \sin \theta=-\omega_{0}^{2} \sin \theta
$$

## Derivation of Equations of Motion

- $m=$ pendulum mass

- $\mathrm{m}_{\text {spring }}=$ spring mass
- I = unstreatched spring length
- $\mathrm{k}=$ spring constant
- $g$ = acceleration due to gravity
- $F_{t}=$ pre-tension of spring
- $r_{s}=$ static spring stretch, $r_{s}=\frac{m g-F_{t}}{k}$
- $r_{d}=$ dynamic spring stretch
- $r=$ total spring stretch $r_{s}+r_{d}$


## Derivation of Equations of Motion -Polar Coordinates



- $\overrightarrow{\mathrm{r}}=\mathrm{r} \hat{\mathrm{e}}_{\mathrm{t}}$
- $\overrightarrow{\mathrm{v}}=\frac{d \vec{r}}{d t}=\dot{\mathrm{r}} \widehat{\mathrm{e}}_{\mathrm{r}}+\mathrm{r} \dot{\theta} \widehat{\theta_{\theta}}=v_{\mathrm{r}} \widehat{\widehat{\mathrm{e}}_{\mathrm{r}}}+\mathrm{v}_{\theta} \widehat{\mathrm{e}_{\theta}}$
- $\vec{a}=\frac{d \vec{v}}{d t}=\left(\ddot{r}-r \dot{\theta^{2}}\right) \widehat{e}_{r}+(r \ddot{\theta}+2 \dot{r} \dot{\theta}) \widehat{e_{\theta}}+a_{r} \widehat{e_{r}}+a_{\theta} \widehat{e_{\theta}}$
- $\mathrm{v}_{\mathrm{r}}\left\{\begin{array}{lr}\text { magnitude change } & \ddot{\mathrm{r}} \\ \text { direction change } & \dot{\mathrm{r}} \dot{\theta}\end{array}\right.$
- $\mathrm{v}_{\theta}\left\{\begin{array}{lr}\text { magnitude change } & \mathrm{r} \ddot{\theta}+\dot{\mathrm{r}} \dot{\theta} \\ \text { direction change } & \mathrm{r} \dot{\theta}^{2}\end{array}\right.$


## Derivation of Equations of Motion -Rigid Body Kinematics



$$
\begin{aligned}
& {\left[\begin{array}{l}
\mathrm{x} \\
\mathrm{y} \\
\mathrm{z}
\end{array}\right]=\left[\begin{array}{ccc}
\cos \theta & \sin \theta & 0 \\
-\sin \theta & \cos \theta & 0 \\
0 & 0 & 1
\end{array}\right]\left[\begin{array}{l}
\mathrm{X} \\
\mathrm{Y} \\
\mathrm{Z}
\end{array}\right]} \\
& {\left[\begin{array}{l}
\hat{1} \\
\hat{\jmath} \\
\hat{\mathrm{k}}
\end{array}\right]=\left[\begin{array}{ccc}
\cos \theta & \sin \theta & 0 \\
-\sin \theta & \cos \theta & 0 \\
0 & 0 & 1
\end{array}\right]\left[\begin{array}{l}
\hat{\mathrm{I}} \\
\hat{\mathrm{~J}} \\
\widehat{\mathrm{~K}}
\end{array}\right]}
\end{aligned}
$$

## Derivation of Equations of Motion -Rigid Body Kinematics

$$
\begin{aligned}
& { }^{\mathrm{R}} \overrightarrow{\mathrm{a}}^{\mathrm{P}}={ }^{\mathrm{R}} \overrightarrow{\mathrm{a}}^{\mathrm{O}}+\left[{ }^{\mathrm{R}} \vec{\omega}^{\mathrm{R}_{1}} \times\left({ }^{\mathrm{R}} \vec{\omega}^{\mathrm{R}_{1}} \times \overrightarrow{\mathrm{r}}^{\mathrm{OP}}\right)\right]+\left[{ }^{\mathrm{R}} \vec{\alpha}^{\mathrm{R}_{1}} \times \overrightarrow{\mathrm{r}}^{\mathrm{OP}}\right]+{ }^{\mathrm{R}_{1}} \vec{a}^{\mathrm{P}}+2\left[{ }^{\mathrm{R}} \vec{\omega}^{\mathrm{R}_{1}} \times{ }^{\mathrm{R}_{1}} \overrightarrow{\mathrm{~V}}^{\mathrm{P}}\right] \\
& { }^{\mathrm{R}} \overrightarrow{\mathrm{a}}^{\mathrm{O}}=0 \\
& { }^{\mathrm{R}} \vec{\omega}^{\mathrm{R}_{1}}=\dot{\theta} \hat{\mathrm{K}}=\dot{\theta} \hat{\mathrm{K}} \\
& \overrightarrow{\mathrm{r}}^{\mathrm{OP}}=-(\ell+\mathrm{r}) \hat{\mathrm{j}}=-(\ell+\mathrm{r})[-\sin \theta \hat{\mathrm{I}}+\cos \theta \hat{\mathrm{J}}] \\
& { }^{\mathrm{R}} \vec{\alpha}^{\mathrm{R}_{1}}=\ddot{\theta} \hat{\mathrm{k}}=\ddot{\theta} \hat{\mathrm{K}} \\
& { }^{\mathrm{R}_{1}} \overrightarrow{\mathrm{~V}}^{\mathrm{P}}=-\hat{\dot{\mathrm{j}}}=-\dot{\mathrm{r}}[-\sin \theta \hat{\mathrm{I}}+\cos \theta \hat{\mathrm{J}}] \\
& { }^{\mathrm{R}_{1}} \vec{a}^{\mathrm{P}}=-\hat{\dot{\mathrm{r}}}=-\ddot{\mathrm{r}}[-\sin \theta \hat{\mathrm{I}}+\cos \theta \hat{\mathrm{J}}]
\end{aligned}
$$

After substitutions and evaluation:


Free Body Diagram

$$
\begin{aligned}
& \sum \mathrm{F}_{\mathrm{r}}=\mathrm{ma}_{\mathrm{r}}=\mathrm{m}\left[\ddot{\mathrm{r}}-(\ell+\mathrm{r}) \dot{\theta}^{2}\right] \\
& \sum \mathrm{F}_{\mathrm{\theta}}=\mathrm{ma}_{\mathrm{\theta}}=\mathrm{m}[(\ell+\mathrm{r}) \ddot{\theta}+2 \dot{\mathrm{i}}]
\end{aligned}
$$

$\mathrm{mir}-\mathrm{m}(\ell+\mathrm{r}) \theta^{2}+\mathrm{kr}+\mathrm{F}_{\mathrm{t}}-\mathrm{mg} \cos \theta=0$ $(\ell+\mathrm{r}) \ddot{\theta}+2 \mathrm{i} \dot{\theta}+\mathrm{g} \sin \theta=0$

$$
{ }^{\mathrm{R}} \overrightarrow{\mathrm{a}}^{\mathrm{P}}=\hat{\mathrm{i}}[(\ell+\mathrm{r}) \ddot{\theta}+2 \dot{\mathrm{r}} \dot{\theta}]+\hat{\mathrm{j}}\left[-\ddot{\mathrm{r}}+(\ell+\mathrm{r}) \dot{\theta}^{2}\right]
$$

## Derivation of Equations of Motion -Lagrange Equations

$$
\begin{aligned}
& T=\frac{1}{2} m\left(\dot{x}^{2}+(\ell+x)^{2} \dot{\theta}^{2}\right) . \quad \text { Kinetic Energy } \\
& V(x, \theta)=-m g(\ell+x) \cos \theta+\frac{1}{2} k x^{2} . \quad \text { Potential Energy } \\
& L \equiv T-V=\frac{1}{2} m\left(\dot{x}^{2}+(\ell+x)^{2} \dot{\theta}^{2}\right)+m g(\ell+x) \cos \theta-\frac{1}{2} k x^{2} .
\end{aligned}
$$

# Derivation of Equations of Motion <br> -Lagrange Equations 

- Lagrange's Equation, Nonlinear equations of motion

$$
\begin{aligned}
\frac{d}{d t}\left(\frac{\partial L}{\partial \dot{x}}\right)=\frac{\partial L}{\partial x} & \Longrightarrow \quad m \ddot{x}=m(\ell+x) \dot{\theta}^{2}+m g \cos \theta-k x \\
\frac{d}{d t}\left(\frac{\partial L}{\partial \dot{\theta}}\right)=\frac{\partial L}{\partial \theta} & \Longrightarrow \quad \frac{d}{d t}\left(m(\ell+x)^{2} \dot{\theta}\right)=-m g(\ell+x) \sin \theta \\
& \Longrightarrow \quad \begin{array}{l}
m(\ell+x)^{2} \ddot{\theta}+2 m(\ell+x) \dot{x} \dot{\theta}=-m g(\ell+x) \sin \theta \\
\\
\end{array} \quad \Longrightarrow \quad m(\ell+x) \ddot{\theta}+2 m \dot{x} \dot{\theta}=-m g \sin \theta
\end{aligned}
$$

## Elastic pendulum in the real world

Pendulum... but not elastic:


Elastic... but not pendulum:


Elastic pendulum in the real world - Spring Swinging


Elastic pendulum in the real world
-Bungee Jumping


## Trivial Cases

- System not integrable
- Initial condition without elastic potential
- Only vertical oscillation
- Initial condition with elastic potential


## Equilibrium States

- Hook's Law $F_{e}=-k\left(l-l_{0}\right)$
- Gravitational Force $F_{g}=m g$
- $\sum F=F_{c}+F_{g}=0$
- At equilibrium $\sum_{k\left(I-t_{0}\right)=m g}$
- System at equilibrium

$$
\begin{aligned}
& \ddot{x}=0 \Rightarrow x=0 \\
& \ddot{y}=0 \Rightarrow y=0 \\
& \ddot{z}=0 \Rightarrow-\frac{k}{m}\left(\frac{l-l_{0}}{l}\right) z-g=0 \Rightarrow z=-l \\
& \text { Stable state } \quad(x, y, z)=(0,0,-l) \\
& \text { Unstable at } \quad(x, y, z)=\left(0,0,2 l_{0}-l\right)
\end{aligned}
$$

## Turning things into a handy system:

$$
\begin{aligned}
& \ddot{x}=-\frac{\omega_{z}^{2}\left(r-l_{0}\right)}{r} x \\
& \ddot{y}=-\frac{\omega_{z}^{2}\left(r-l_{0}\right)}{r} y \\
& \ddot{z}=-\frac{\omega_{z}^{2}\left(r-l_{0}\right)}{r} z-g
\end{aligned}
$$

Where $\quad \omega_{z}=\sqrt{\frac{k}{m}} \quad$ and $\quad r=\sqrt{x^{2}+y^{2}+z^{2}}$

## Some regimes make familiar shapes!



## What happens if we shake things up?



## Turning things up a bit...



## Let's take a closer look at the same regime:



## Now look at the XY plane again.

Is it the swivel that is causing the pendulum to avoid the center?


Awesome they almost meet! (Quasi-awesome)


## Dr. Peter Lynch's model:

Initial Conditions:
$x 0=0.01$;
xdot0=0.00;
y0=0.00;
ydot0=0.02;
zprime0=0.1;
zdot0=0.00;


## References (1/2)

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