Hanging Chain Vibration Modes MATH 485: Mid-Term Presentation

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Outline

Model

- Continuum Chain
- N-Pendulum





Figure: Hanging Chains [2]018

Hanging Chain Vibration Modes

Outline



Continuum Chain

N-Pendulum



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Hanging Chain Classical Model

Governing Equation

Model, u(x, t)

- u Transverse displacement [m]
- × Vertical height [m]
- t Time [*s*]
- ν Linear Density $\left[\frac{kg}{m}\right]$
- g Gravitational Acceleration $\left[\frac{m}{s^2}\right]$

Boundary Condition

• $u(L,t) \equiv 0$

Assumptions

- Small Oscillations.
- Uniform Density.





Hanging Chain Classical Model

Governing Equation

Observation

$$\mathbf{T}(x) \cdot \hat{y} = \nu g x \tan(\theta) \approx \nu g x \frac{\partial u}{\partial x}$$

Apply Newton's Third Law, $\sum \mathbf{F} \cdot \hat{y} = m\mathbf{a} \cdot \hat{y}$



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Hanging Chain Governing Equation

Governing Equation

 $u_{tt} = g(u_x + x u_{xx})$

Let
$$X = \frac{x}{L}$$
, $U = \frac{u}{L}$, $au = \frac{t}{\sqrt{\frac{g}{L}}}$.
Thus, $X \in [0,1]$, $U \in [0,1]$, $\sqrt{\frac{g}{L}}$ characteristic time

Non-Dimensionalized Governing Equation $U_{\tau\tau} = U_X + XU_{XX}$ $U(1, \tau) \equiv 0$

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Hanging Chain Classical Solution

Governing Equation* $U_{tt} = U_x + xU_{xx}$ $u(1, t) \equiv 0$

Apply Separation of Variables. Let $U(X, \tau) = X(x) \cdot T(t)$, thus

$$\frac{T''}{T} = \frac{X' + x \cdot X''}{X} = -\lambda^2 \qquad \lambda \in \mathbb{R}$$

Let $z^2 = 4x$, use **chain** rule,

$$\begin{cases} \mathsf{T}'' + \lambda^2 T = 0\\ \mathsf{z}^2 X'' + \mathsf{z} X' + \mathsf{z}^2 \lambda^2 X = 0 \end{cases}$$

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Hanging Chain Classical Solution

Governing Equations*

$$\begin{cases} T'' + \lambda^2 T = 0\\ z^2 X'' + z X' + z^2 \lambda^2 X = 0 \end{cases}$$

Time Oscillation $\begin{cases} \sin(|\lambda|t) \\ \cos(|\lambda|t) \end{cases}$

Spacial ODE is well known, Bessel Equation of zeroth order with solutions of Bessel function of the first and second kind

Spacial Profile,
$$\begin{cases} J_0(\lambda z) = J_0(2\lambda\sqrt{x}) \\ Y_0(\lambda z) = Y_0(2\lambda\sqrt{x})) \end{cases}$$

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Continuum Chain

Hanging Chain Bessel Function

 $Y_0(x)$ has an asymptote at x = 0.



Figure: $J_0(x)$, $Y_0(x)$ [3]

Hanging Chain Superposition

By linearity, $u(x,t) = X(x) \cdot T(t)$ is,

$$u(x,t) = \sum_{n=0}^{\infty} \left\{ A_n \sin(\lambda_n t) + B_n \cos(\lambda_n t) \right\} J_0(2\lambda_n \sqrt{x})$$

Recall the boundary condition, u(1, t) = 0,

$$\sum_{n=0}^{\infty} \left\{ A_n \sin(\lambda_n t) + B_n \cos(\lambda_n t) \right\} J_0(2\lambda_n \sqrt{x}) = 0$$

For non-trivial result, A_n , $B_n \neq 0$, thus $J_0(2\lambda_n) = 0$, e.g. roots of J_0 determine λ_n .

Harmonic Frequencies

Figure: Modes 1-3, first three Bessel zeros [1]

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Outline



• Continuum Chain

N-Pendulum

Other Literature

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Discrete Model, $u_i(t)$.

- u Transverse displacement [m]
- t Time [*s*]
- g Gravitational Acceleration $\left[\frac{m}{s^2}\right]$

Assumptions

- Small Oscillations.
- Equal

Length and Mass, e.g. $m_i = m_j$ $l_q = l_p \forall i, j, p, q$



Figure: N-Pendulum [4].

N-Pendulum

Governing Equations

 $\sum \mathbf{F} = m\mathbf{a}$, for small oscillations, motion in y direction is negligible, and $tan(\theta) \approx \theta$

$$F_i - F_{i-1} = m_i \ddot{u}_i$$

$$\sum_{j=i+1}^{n} m_j gtan(\theta_i) - \sum_{j=i}^{n} m_j gtan(\theta_{i-1}) = m_i \ddot{u}_i$$
$$(n-i)g\theta_i - (n-i+1)g\theta_{i-1} = \ddot{x}$$

Using the small angle approximation, $\theta_i \approx \frac{u_i}{l}$, we find,

$$\ddot{u}_i = \frac{g}{l}[(n-i)(u_{i+1}-2u_i+u_{i-1})-u_i+u_i]$$



Figure: N-Pendulum [4].

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N-Pendulum

Governing Equations

N-Pendulum Governing Equation

$$\ddot{U}_i = \frac{g}{l}[(n-i)(u_{i+1}-2u_i+u_{i-1})-u_i+u_i]$$

Let
$$U = rac{u}{nL}$$
, $au = rac{t}{\sqrt{rac{g}{L}}}$.
Thus $U \in [0,1]$, $\sqrt{rac{g}{L}}$ characteristic time

N-Pendulum Governing Equation* $U_{\tau\tau} = (n-i)(U_{i+1} - 2U_i + U_{i-1}) - U_i + U_i$

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N-Pendulum

Harmonic Frequencies

N-Pendulum Governing Equation** $\ddot{U} = (n - i)(U_{i+1} - 2U_i + U_{i-1}) - U_i + U_i$

In matrix form, $\frac{d^2}{dt^2}\mathbf{U} = \mathbf{AU}$, where, **A** is given as,

$$\mathbf{A} = \begin{bmatrix} 1-2n & n-1 & \dots & \\ n-1 & 3-2n & n-2 & \dots & \\ & n-2 & 5-2n & n-3 & \dots & \\ \vdots & \vdots & \vdots & \ddots & \vdots & \\ & & & 2 & -3 & 1 \\ & & & & 1 & -1 \end{bmatrix}$$

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N-Pendulum

Harmonic Frequencies

N-Pendulum Matrix Equation

 $\frac{d^2}{dt^2}\mathbf{U} = \mathbf{AU}$

Let **U** take a resonant form of $\mathbf{U} = \mathbf{v}e^{\omega_i t}$. Thus, $\frac{d^2}{dt^2}\mathbf{U} = \omega_i^2\mathbf{U}$. Thus,

$$(\mathbf{A} - \omega_i^2 \mathbb{1})\mathbf{U} = \mathbf{0}$$

For non-trivial solution, we solve,

$$\det(\mathbf{A} - \omega_i^2 \mathbb{1}) = 0$$

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N-Pendulum

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https://www.youtube.com/watch?v=XVIi7-u9wI4 [10]
https://youtu.be/-ztU56yG6aY
https://youtu.be/4FrdYwOIbjw
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Model

N-Pendulum

N-Pendulum



Figure: N=20 Pendulum Simulation [10]

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Other Literature

- Concentrated tip-mass
 - T(x) = mg + gx, same equation up to translation
 - New Boundary Conditions
- Control Systems
 - Flatness
 - Spring Feedback



Summary

- The classical hanging chain results in zeroth order first-kind Bessel function, *J*₀, standing waves.
- Natural frequencies are determined by zeroes of J_0 .
- The $\lim_{n\to\infty}$ *n*-pendulum approaches similar behavior.
- Outlook
 - Verify Model
 - Numerically, Experimentally, Analytically
 - Introduce Complications
 - Air Drag
 - Non-uniform Density
 - Driven System (water hose)
 - Sliding Chain
 - Control Theory Stabilization
 - Magnetic Chains [6]

For Further Reading I

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- 📎 S. Weibel ; J. Baillieul ; B. Lehman, "Equilibria and stability of an n-pendulum forced by rapid oscillations," Decision and Control, 1997 http://ieeexplore.ieee.org/abstract/document/657602/?reloa

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For Further Reading

For Further Reading II

- S. Weibel; J. Baillieul; B. Lehman, "Stability of vertical magnetic chains," The Royal Society, 2017 http://rspa.royalsocietypublishing.org/content/473/2198/20
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- Boundary Conditions and Modes of the Vertically Hanging Chain Y. Verbin, 2014 arXiv:1412.1846 [physics.class-ph] https://arxiv.org/pdf/1412.1846.pdf

For Further Reading III

- 🌑 C. Y. Wang C. M. Wang (2010) Exact solutions for vibration of a vertical heavy string with a tip mass, The IES Journal Part A: Civil Structural Engineering, 3:4, 278-281, DOI: 10.1080/19373260.2010.521623 https://www.tandfonline.com/doi/pdf/10.1080/19373260.2010.
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